

16.20 HANDOUT #2

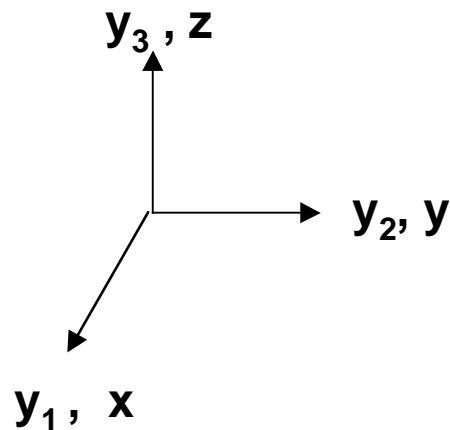
Fall, 2002

Review of General Elasticity

NOTATION REVIEW (e.g., for strain)

<u>Engineering</u>		<u>Contracted</u>		<u>Engineering "Tensor"</u>		<u>Tensor</u>
ϵ_x	=	ϵ_1	=	ϵ_{xx}	=	ϵ_{11}
ϵ_y	=	ϵ_2	=	ϵ_{yy}	=	ϵ_{22}
ϵ_z	=	ϵ_3	=	ϵ_{zz}	=	ϵ_{33}
γ_{yz}	=	ϵ_4	=	$2 \epsilon_{yz}$	=	$2 \epsilon_{23}$
γ_{xz}	=	ϵ_5	=	$2 \epsilon_{xz}$	=	$2 \epsilon_{13}$
γ_{xy}	=	ϵ_6	=	$2 \epsilon_{xy}$	=	$2 \epsilon_{12}$

EQUATIONS OF ELASTICITY



15 equations/15 unknowns

Right-handed rectangular Cartesian coordinate system

1. Equilibrium (3)

$$\left. \begin{aligned}
 \frac{\partial \sigma_{11}}{\partial y_1} + \frac{\partial \sigma_{21}}{\partial y_2} + \frac{\partial \sigma_{31}}{\partial y_3} + f_1 &= 0 \\
 \frac{\partial \sigma_{12}}{\partial y_1} + \frac{\partial \sigma_{22}}{\partial y_2} + \frac{\partial \sigma_{32}}{\partial y_3} + f_2 &= 0 \\
 \frac{\partial \sigma_{13}}{\partial y_1} + \frac{\partial \sigma_{23}}{\partial y_2} + \frac{\partial \sigma_{33}}{\partial y_3} + f_3 &= 0
 \end{aligned} \right\} \frac{\partial \sigma_{mn}}{\partial y_m} + f_n = 0$$

2. Strain-Displacement (6)

$$\left. \begin{aligned} \varepsilon_{11} &= \frac{\partial u_1}{\partial y_1} & \varepsilon_{21} = \varepsilon_{12} &= \frac{1}{2} \left(\frac{\partial u_1}{\partial y_2} + \frac{\partial u_2}{\partial y_1} \right) \\ \varepsilon_{22} &= \frac{\partial u_2}{\partial y_2} & \varepsilon_{31} = \varepsilon_{13} &= \frac{1}{2} \left(\frac{\partial u_1}{\partial y_3} + \frac{\partial u_3}{\partial y_1} \right) \\ \varepsilon_{33} &= \frac{\partial u_3}{\partial y_3} & \varepsilon_{32} = \varepsilon_{23} &= \frac{1}{2} \left(\frac{\partial u_2}{\partial y_3} + \frac{\partial u_3}{\partial y_2} \right) \end{aligned} \right\} \varepsilon_{mn} = \frac{1}{2} \left(\frac{\partial u_m}{\partial y_n} + \frac{\partial u_n}{\partial y_m} \right)$$

3. Stress-Strain (6)

Generalized Hooke's Law: $\sigma_{mn} = E_{mnpq} \varepsilon_{pq}$

- Anisotropic:

$$\begin{Bmatrix} \sigma_{11} \\ \sigma_{22} \\ \sigma_{33} \\ \sigma_{23} \\ \sigma_{13} \\ \sigma_{12} \end{Bmatrix} = \begin{bmatrix} E_{1111} & E_{1122} & E_{1133} & 2E_{1123} & 2E_{1113} & 2E_{1112} \\ E_{1122} & E_{2222} & E_{2233} & 2E_{2223} & 2E_{2213} & 2E_{2212} \\ E_{1133} & E_{2233} & E_{3333} & 2E_{3323} & 2E_{3313} & 2E_{3312} \\ E_{1123} & E_{2223} & E_{3323} & 2E_{2323} & 2E_{1323} & 2E_{1223} \\ E_{1113} & E_{2213} & E_{3313} & 2E_{1323} & 2E_{1313} & 2E_{1213} \\ E_{1112} & E_{2212} & E_{3312} & 2E_{1223} & 2E_{1213} & 2E_{1212} \end{bmatrix} \begin{Bmatrix} \varepsilon_{11} \\ \varepsilon_{22} \\ \varepsilon_{33} \\ \varepsilon_{23} \\ \varepsilon_{13} \\ \varepsilon_{12} \end{Bmatrix}$$

- Orthotropic:

$$\begin{Bmatrix} \sigma_{11} \\ \sigma_{22} \\ \sigma_{33} \\ \sigma_{23} \\ \sigma_{13} \\ \sigma_{12} \end{Bmatrix} = \begin{bmatrix} E_{1111} & E_{1122} & E_{1133} & 0 & 0 & 0 \\ E_{1122} & E_{2222} & E_{2233} & 0 & 0 & 0 \\ E_{1133} & E_{2233} & E_{3333} & 0 & 0 & 0 \\ 0 & 0 & 0 & 2E_{2323} & 0 & 0 \\ 0 & 0 & 0 & 0 & 2E_{1313} & 0 \\ 0 & 0 & 0 & 0 & 0 & 2E_{1212} \end{bmatrix} \begin{Bmatrix} \varepsilon_{11} \\ \varepsilon_{22} \\ \varepsilon_{33} \\ \varepsilon_{23} \\ \varepsilon_{13} \\ \varepsilon_{12} \end{Bmatrix}$$

Compliance Form: $\varepsilon_{mn} = S_{mnpq} \sigma_{pq}$

where: $\tilde{E}^{-1} = \tilde{S}$

DEFINITION OF ENGINEERING CONSTANTS

1. Longitudinal (Young's) (Extensional) Moduli:

$$E_{mm} = \frac{\sigma_{mm}}{\varepsilon_{mm}} \quad \text{due to } \sigma_{mm} \text{ applied } \underline{\text{only}} \quad (\text{no summation on } m)$$

2. Poisson's Ratios:

$$\nu_{nm} = -\frac{\varepsilon_{mm}}{\varepsilon_{nn}} \quad \text{due to } \sigma_{nn} \text{ applied } \underline{\text{only}} \quad (\text{for } n \neq m)$$

$$\text{Reciprocity: } \nu_{nm} E_m = \nu_{mn} E_n \quad (\text{no sum}) \\ (m \neq n)$$

3. Shear Moduli:

$$G_{mn} = \frac{\sigma_{mn}}{2\varepsilon_{mn}} \quad \text{due to } \sigma_{mn} \text{ applied } \underline{\text{only}} \\ (\text{for } m = 4, 5, 6) \\ (\text{for } n \neq m)$$

4. Coefficients of Mutual Influence: (using contracted notation)

$$\eta_{mn} = \frac{-\varepsilon_n}{\varepsilon_m} \quad \text{for } \sigma_m \text{ applied } \underline{\text{only}} \\ (\text{for } m, n, = 1, 2, 3, 4, 5, 6, m \neq n)$$

(Note: one strain extensional, one strain shear)

Reciprocity here as well

5. Chentsov Coefficients: (using contracted notation)

$$\eta_{mn} = \frac{-\varepsilon_n}{\varepsilon_m} \quad \text{for } \sigma_m \text{ applied } \underline{\text{only}} \\ (\text{for } m, n, = 4, 5, 6, m \neq n)$$

“ENGINEERING” STRESS-STRAIN EQUATIONS (*using contracted notation*)

$$\varepsilon_1 = \frac{1}{E_1} [\sigma_1 - \nu_{12}\sigma_2 - \nu_{13}\sigma_3 - \eta_{14}\sigma_4 - \eta_{15}\sigma_5 - \eta_{16}\sigma_6]$$

$$\varepsilon_2 = \frac{1}{E_2} [-\nu_{21}\sigma_1 + \sigma_2 - \nu_{23}\sigma_3 - \eta_{24}\sigma_4 - \eta_{25}\sigma_5 - \eta_{26}\sigma_6]$$

$$\varepsilon_3 = \frac{1}{E_3} [-\nu_{31}\sigma_1 - \nu_{32}\sigma_2 + \sigma_3 - \eta_{34}\sigma_4 - \eta_{35}\sigma_5 - \eta_{36}\sigma_6]$$

$$\gamma_4 = \varepsilon_4 = \frac{1}{G_4} [-\eta_{41}\sigma_1 - \eta_{42}\sigma_2 - \eta_{43}\sigma_3 + \sigma_4 - \eta_{45}\sigma_5 - \eta_{46}\sigma_6]$$

$$\gamma_5 = \varepsilon_5 = \frac{1}{G_5} [-\eta_{51}\sigma_1 - \eta_{52}\sigma_2 - \eta_{53}\sigma_3 - \eta_{54}\sigma_4 + \sigma_5 - \eta_{56}\sigma_6]$$

$$\gamma_6 = \varepsilon_6 = \frac{1}{G_6} [-\eta_{61}\sigma_1 - \eta_{62}\sigma_2 - \eta_{63}\sigma_3 - \eta_{64}\sigma_4 - \eta_{65}\sigma_5 + \sigma_6]$$

In general:

$$\varepsilon_n = -\frac{1}{E_n} \sum_{m=1}^6 \nu_{nm} \sigma_m$$

Note: $\nu_{nn} = -1$
and η 's \rightarrow ν 's

• Orthotropic form

In terms of ENGINEERING CONSTANTS (*using contracted notation*):

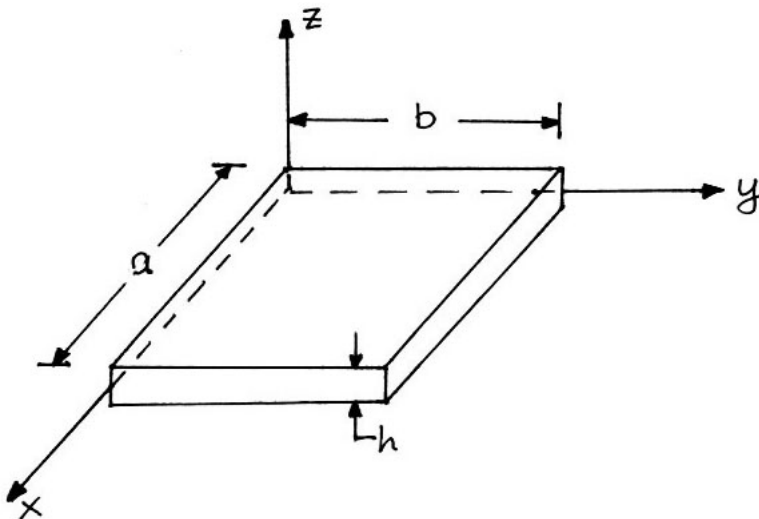
$$\begin{Bmatrix} \varepsilon_1 \\ \varepsilon_2 \\ \varepsilon_3 \\ \varepsilon_4 \\ \varepsilon_5 \\ \varepsilon_6 \end{Bmatrix} = \begin{bmatrix} \frac{1}{E_1} & -\frac{\nu_{12}}{E_1} & -\frac{\nu_{13}}{E_1} & 0 & 0 & 0 \\ -\frac{\nu_{21}}{E_2} & \frac{1}{E_2} & -\frac{\nu_{23}}{E_2} & 0 & 0 & 0 \\ -\frac{\nu_{31}}{E_3} & -\frac{\nu_{32}}{E_3} & \frac{1}{E_3} & 0 & 0 & 0 \\ 0 & 0 & 0 & \frac{1}{G_4} & 0 & 0 \\ 0 & 0 & 0 & 0 & \frac{1}{G_5} & 0 \\ 0 & 0 & 0 & 0 & 0 & \frac{1}{G_6} \end{bmatrix} \begin{Bmatrix} \sigma_1 \\ \sigma_2 \\ \sigma_3 \\ \sigma_4 \\ \sigma_5 \\ \sigma_6 \end{Bmatrix}$$

• Isotropic form

$$\begin{Bmatrix} \varepsilon_1 \\ \varepsilon_2 \\ \varepsilon_3 \\ \varepsilon_4 \\ \varepsilon_5 \\ \varepsilon_6 \end{Bmatrix} = \begin{bmatrix} 1/E & -\nu/E & -\nu/E & 0 & 0 & 0 \\ -\nu/E & 1/E & -\nu/E & 0 & 0 & 0 \\ -\nu/E & -\nu/E & 1/E & 0 & 0 & 0 \\ 0 & 0 & 0 & 1/G & 0 & 0 \\ 0 & 0 & 0 & 0 & 1/G & 0 \\ 0 & 0 & 0 & 0 & 0 & 1/G \end{bmatrix} \begin{Bmatrix} \sigma_1 \\ \sigma_2 \\ \sigma_3 \\ \sigma_4 \\ \sigma_5 \\ \sigma_6 \end{Bmatrix}$$

with: $G = \frac{E}{2(1 + \nu)}$

PLANE STRESS



$$h \ll a, b$$

$$\sigma_{zz}, \sigma_{yz}, \sigma_{xz} = 0$$

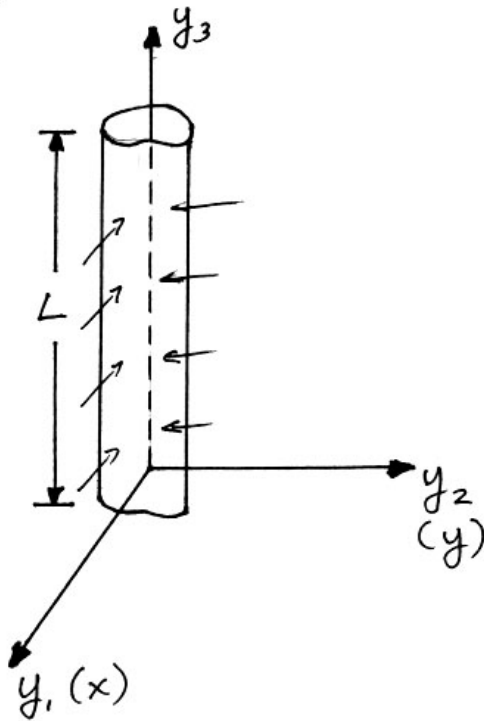
$$\frac{\partial}{\partial z} = 0$$

Anisotropic stress-strain equations

$$\left. \begin{aligned} \varepsilon_1 &= \frac{1}{E_1} [\sigma_1 - \nu_{12}\sigma_2 - \eta_{16}\sigma_6] \\ \varepsilon_2 &= \frac{1}{E_2} [-\nu_{21}\sigma_1 + \sigma_2 - \eta_{26}\sigma_6] \\ \varepsilon_6 &= \frac{1}{G_6} [-\eta_{61}\sigma_1 - \eta_{62}\sigma_2 + \sigma_6] \end{aligned} \right\} \text{Primary}$$

$$\left. \begin{aligned}
 \varepsilon_3 &= \frac{1}{E_3} [-\nu_{31}\sigma_1 - \nu_{32}\sigma_2 - \eta_{36}\sigma_6] \\
 \varepsilon_4 &= \frac{1}{G_4} [-\eta_{41}\sigma_1 - \eta_{42}\sigma_2 - \eta_{46}\sigma_6] \\
 \varepsilon_5 &= \frac{1}{G_5} [-\eta_{51}\sigma_1 - \eta_{52}\sigma_2 - \eta_{56}\sigma_6]
 \end{aligned} \right\} \text{Secondary}$$

PLANE STRAIN



$$L \gg x, y$$

$$\frac{\partial}{\partial z} = 0$$

$$\varepsilon_{13} = \varepsilon_{23} = \varepsilon_{33} = 0$$

SUMMARY

	<u>Plane Stress</u>	<u>Plane Strain</u>
Geometry:	thickness (y_3) \ll in-plane dimensions (y_1, y_2)	length (y_3) \gg in-plane dimensions (y_1, y_2)
Loading:	$\sigma_{33} \ll \sigma_{\alpha\beta}$	$\sigma_{\alpha\beta}$ only $\partial/\partial y_3 = 0$
Resulting Assumptions:	$\sigma_{i3} = 0$	$\varepsilon_{i3} = 0$
Primary Variables:	$\varepsilon_{\alpha\beta}, \sigma_{\alpha\beta}, u_\alpha$	$\varepsilon_{\alpha\beta}, \sigma_{\alpha\beta}, u_\alpha$
Secondary Variable(s):	ε_{33}, u_3	σ_{33}
Note:	Eliminate ε_{33} from eq. set by using $\sigma_{33} = 0$ $\sigma - \varepsilon$ eq. and expressing ε_{33} in terms of $\varepsilon_{\alpha\beta}$	Eliminate σ_{33} from eq. Set by using σ_{33} $\sigma - \varepsilon$ eq. and expressing σ_{33} in terms of $\varepsilon_{\alpha\beta}$

TRANSFORMATIONS

$$\tilde{\sigma}_{mn} = l_{\tilde{m}p} l_{\tilde{n}q} \sigma_{pq}$$

$$\tilde{\varepsilon}_{mn} = l_{\tilde{m}p} l_{\tilde{n}q} \varepsilon_{pq}$$

$$\tilde{x}_m = l_{\tilde{m}p} x_p$$

$$\tilde{u}_m = l_{\tilde{m}p} u_p$$

$$\tilde{E}_{mnpq} = l_{\tilde{m}r} l_{\tilde{n}s} l_{\tilde{p}t} l_{\tilde{q}u} E_{rstu}$$

where: $l_{\tilde{m}n}$ = cosine of angle from \tilde{y}_m to y_n

OTHER COORDINATE SYSTEMS

$$F_1 (y_1, y_2, y_3) = \xi$$

$$F_2 (y_1, y_2, y_3) = \eta$$

$$F_3 (y_1, y_2, y_3) = \zeta$$

Example - Cylindrical Coordinates

$$\xi = r \quad F_1 (y_1, y_2, y_3) = \sqrt{y_1^2 + y_2^2}$$

$$\eta = \theta \quad F_2 (y_1, y_2, y_3) = \tan^{-1} (y_2 / y_1)$$

$$\zeta = z \quad F_3 (y_1, y_2, y_3) = y_3$$

• Equilibrium:

$$r: \frac{\partial \sigma_{rr}}{\partial r} + \frac{1}{r} \frac{\partial \sigma_{\theta r}}{\partial \theta} + \frac{\partial \sigma_{zr}}{\partial z} + \frac{\sigma_{rr} - \sigma_{\theta\theta}}{r} + f_r = 0$$

$$\theta: \frac{\partial \sigma_{r\theta}}{\partial r} + \frac{1}{r} \frac{\partial \sigma_{\theta\theta}}{\partial \theta} + \frac{\partial \sigma_{z\theta}}{\partial z} + \frac{2\sigma_{r\theta}}{r} + f_\theta = 0$$

$$z: \frac{\partial \sigma_{rz}}{\partial r} + \frac{1}{r} \frac{\partial \sigma_{\theta z}}{\partial \theta} + \frac{\partial \sigma_{zz}}{\partial z} + \frac{\sigma_{rz}}{r} + f_z = 0$$

• (Engineering) Strain-Displacement:

$$\varepsilon_{rr} = \frac{\partial u_r}{\partial r}$$

$$\varepsilon_{\theta\theta} = \frac{1}{r} \frac{\partial u_\theta}{\partial \theta}$$

$$\varepsilon_{zz} = \frac{\partial u_3}{\partial z}$$

$$\varepsilon_{r\theta} = \frac{\partial u_\theta}{\partial r} + \frac{1}{r} \frac{\partial u_r}{\partial \theta} - \frac{u_\theta}{r}$$

$$\varepsilon_{\theta z} = \frac{1}{r} \frac{\partial u_3}{\partial \theta} + \frac{\partial u_\theta}{\partial z}$$

$$\varepsilon_{zr} = \frac{\partial u_r}{\partial z} + \frac{\partial u_3}{\partial r}$$

• (Isotropic) Stress-Strain:

$$\varepsilon_{rr} = \frac{1}{E} [\sigma_{rr} - \nu(\sigma_{\theta\theta} + \sigma_{zz})]$$

$$\varepsilon_{\theta\theta} = \frac{1}{E} [\sigma_{\theta\theta} - \nu(\sigma_{rr} + \sigma_{zz})]$$

$$\varepsilon_{zz} = \frac{1}{E} [\sigma_{zz} - \nu(\sigma_{rr} + \sigma_{\theta\theta})]$$

$$\varepsilon_{r\theta} = \frac{2(1 + \nu)}{E} \sigma_{r\theta}$$

$$\varepsilon_{\theta z} = \frac{2(1 + \nu)}{E} \sigma_{\theta z}$$

$$\varepsilon_{zr} = \frac{2(1 + \nu)}{E} \sigma_{zr}$$

STRESS FUNCTIONS

$$\nabla^4 \phi = -E\alpha \nabla^2(\Delta T) - (1 - \nu) \nabla^2 V \quad (\text{isotropic})$$

$$\text{where: } \nabla^2 = \frac{\partial^2}{\partial x^2} + \frac{\partial^2}{\partial y^2}$$

$$\sigma_{xx} = \frac{\partial^2 \phi}{\partial y^2} + V$$

$$\sigma_{yy} = \frac{\partial^2 \phi}{\partial x^2} + V$$

$$\sigma_{xy} = -\frac{\partial^2 \phi}{\partial x \partial y}$$

EFFECTS OF THE ENVIRONMENT

Temperature

- Thermal Strain: $\epsilon^T = \alpha \Delta T$

α = Coefficient of Thermal Expansion (C.T.E.)

$$\text{general form: } \epsilon_{ij}^T = \alpha_{ij} \Delta T$$

- Total Strain = Mechanical Strain + Thermal Strain

$$\epsilon_{ij} = \epsilon_{ij}^M + \epsilon_{ij}^T$$

$$\epsilon_{ij}^M = S_{ijkl} \sigma_{kl}$$

- $\sigma_{kl} = E_{ijkl} \epsilon_{ij} - E_{ijkl} \alpha_{ij} \Delta T$

- Transformation of α_{ij} :

$$\left. \begin{aligned} \tilde{\alpha}_{11} &= \cos^2 \theta \alpha_{11}^* + \sin^2 \theta \alpha_{22}^* \\ \tilde{\alpha}_{22} &= \sin^2 \theta \alpha_{11}^* + \cos^2 \theta \alpha_{22}^* \\ \tilde{\alpha}_{12} &= \cos \theta \sin \theta (\alpha_{22}^* - \alpha_{11}^*) \end{aligned} \right\} \alpha_{11}^*, \alpha_{22}^* \text{ are C.T.E.'s in principal material axes}$$

Sources of temperature differential

- Ambient environment
- Convection
- Aerodynamic heating

$$\text{Adiabatic wall temp} = T_{AW} = \left[1 + \frac{\gamma - 1}{2} r M_\infty^2 \right] T_\infty$$

specific heat ratio
Mach number
↓
↓
↑
↑
recovery factor

T_∞ = ambient temperature (°K)

$$\text{heat flux: } q = h (T_{AW} - T_s)$$

↑
heat transfer coefficient

↑
surface temperature of body

- Radiation

- Emissivity

$$q = -\epsilon \sigma T_s^4 \quad \left\{ \begin{array}{l} q = \text{heat flux} \\ \epsilon = \text{emissivity} \\ \sigma = \text{Stefan-Boltzman constant} \\ T_s = \text{surface temperature} \end{array} \right.$$

- Absorptivity

$$q = \alpha I_s \lambda \quad \left\{ \begin{array}{l} q = \text{heat flux} \\ \alpha = \text{absorptivity} \\ I_s = \text{intensity of source} \\ \lambda = \text{angle factor} \end{array} \right.$$

- Conduction

$$q_i^T = -k_{ij}^T \frac{\partial T}{\partial x_j} \quad \left\{ \begin{array}{l} q_i^T = \text{heat flux} \\ k_{ij}^T = \text{thermal conductivity} \end{array} \right.$$

Fourier's equation:

$$\frac{k_z^T}{\rho C} \frac{\partial^2 T}{\partial z^2} = \frac{\partial T}{\partial t}$$

↑
thermal
conductivity

Degradation of material properties

- Glass transition temperature
- $E(T)$, $\sigma_{ult}(T)$, $\sigma_y(T)$
- Creep

Other Environmental Effects

$$\varepsilon_{ij} = \varepsilon_{ij}^M + \sum \varepsilon_{ij}^E$$

total = mechanical + \sum environmental

- Moisture:

$$\varepsilon_{ij}^S = \beta_{ij} C \quad \left\{ \begin{array}{l} \varepsilon_{ij}^S = \text{swelling strain} \\ \beta_{ij} = \text{swelling coefficient} \\ c = \text{moisture concentration} \end{array} \right.$$

- General:

$$\varepsilon_{ij}^E = \chi_{ij} \chi \quad \left\{ \begin{array}{l} \varepsilon_{ij}^E = \text{environmental strain} \\ \chi_{ij} = \text{environmental operator} \\ \chi = \text{environmental scalar} \end{array} \right.$$

Piezoelectricity

- Piezoelectric strain:

$$\varepsilon_{ij}^P = d_{ijk} E_k$$

E_k = electric field

d_{ijk} = piezoelectric constant

- Coupled equations:

$$\sigma_{mn} = E_{mnij} \varepsilon_{ij} - E_{mnij} d_{ijk} E_k$$

$$D_i = e_{ik} E_k + d_{inm} \sigma_{mn}$$

e_{ik} = dielectric constant

D_i = electrical charge